

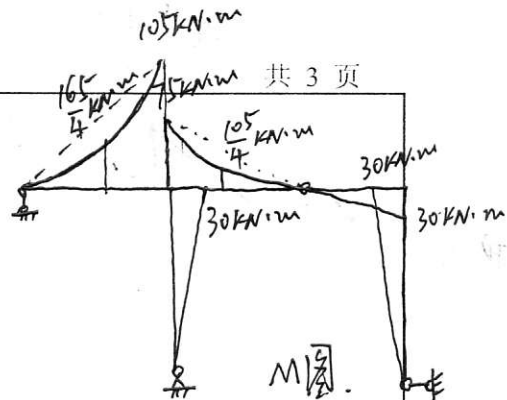
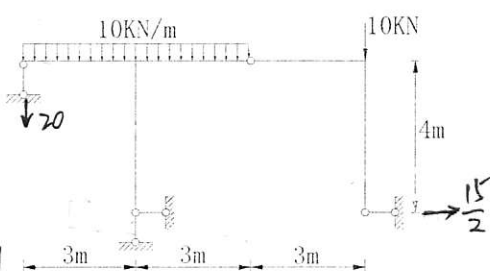
华南理工大学 2000 年攻读硕士学位研究生入学考试试卷

(试卷上做答无效, 请在答题纸上做答, 试后本卷必须与答题纸一同交回)

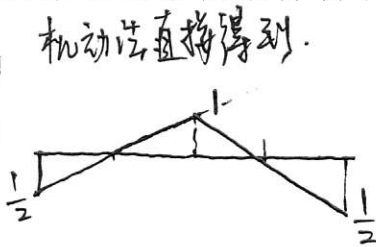
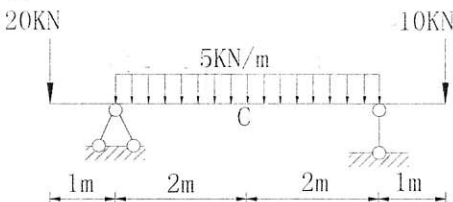
科目名称: 结构力学

适用专业: 工程力学 结构工程

一、作图示结构的 M 图。(12 分)



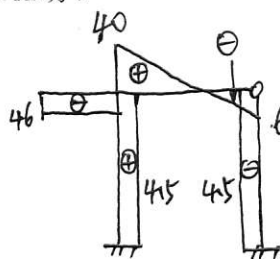
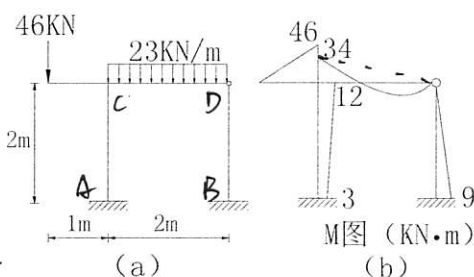
二、作图示梁 M_c 的影响线, 并利用影响线求图示荷载下的 M_c 值。(12 分)



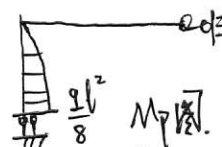
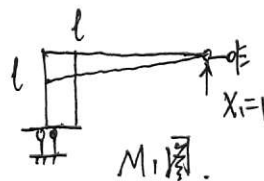
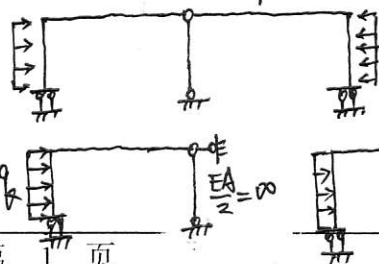
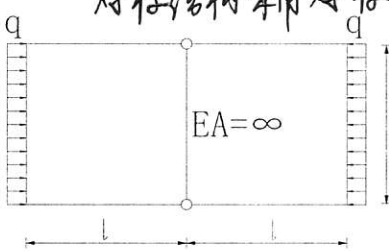
$$M_c = 20 \times \frac{1}{2} + 10 \times \frac{1}{2} + \frac{1}{2} \times 4 \times 1 \times 5 = -5 \text{ kN}\cdot\text{m}$$

(上侧受拉)

三、图 b 是图 a 的 M 图, 试作 Q 图。(12 分)



四、已知 EI = 常数, 试用力法计算并作图示对称结构的 M 图。(15 分)



力法方程: $S_{11} X_1 + \Delta_{1P} = 0$
 $X_1 = -\frac{\Delta_{1P}}{S_{11}}$

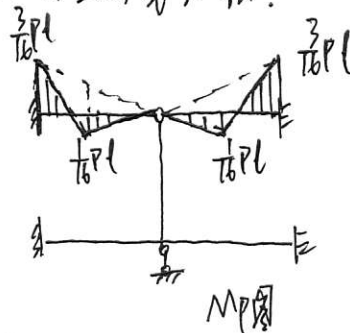
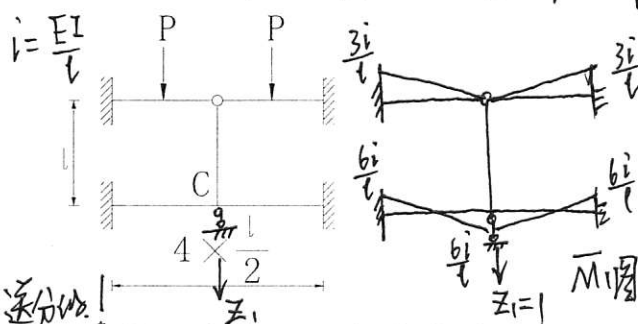
其中: $S_{11} = (\frac{1}{2} \times l \times l + \frac{1}{2} \times l \times \frac{2}{3} l) \cdot \frac{1}{EI} = \frac{5l^3}{6EI}$
 $\Delta_{1P} = \frac{2}{3} \times \frac{1}{2} \times \frac{9l^2}{8} \cdot l \cdot \frac{1}{EI} = \frac{9l^4}{24EI}$

$M = \bar{M}_1 X_1 + M_P$ 作 M 图

(注: 求得的 X_1 为链杆实际轴力的一半)

五、已知图示结构C点线位移为 $Pl^3/48EI$ (\downarrow), EI =常数, 作M图。(12分)

此题不用位移法。C结点并无转角, 只有竖向线位移。



$$M = \bar{M}_1 Z_1 + M_P$$

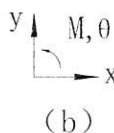
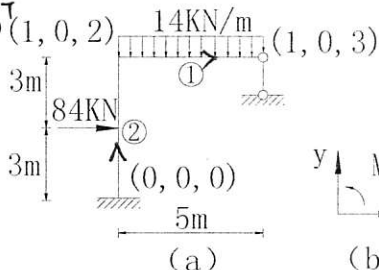
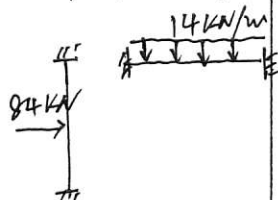
其中 $Z_1 = \frac{Pl^3}{48EI}$ (\downarrow)

可得M图。

六、图a所示结构, 不考虑轴向变形, 整体坐标见图b, 图中圆括号内数码为结点定位向量(力和位移均按水平、竖直、转动方向顺序排列), 求等效结点荷载列阵 $\{P_E\}$ 。(12分)

$$\lambda^{\text{①}} = (1, 0, 2, 1, 0, 3)^T$$

$$\lambda^{\text{②}} = (0, 0, 0, 1, 0, 2)^T$$



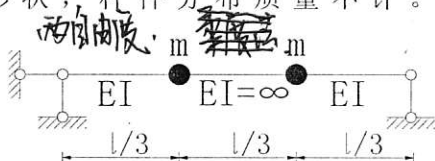
$$F_P^{\text{①}} = (-42, 0, 63, -42, 0, -63)^T$$

$$F_P^{\text{②}} = (0, 35, \frac{175}{6}, 0, 35, -\frac{175}{6})^T$$

$$\{F_P\} = (-42, -\frac{203}{6}, -\frac{175}{6})^T$$

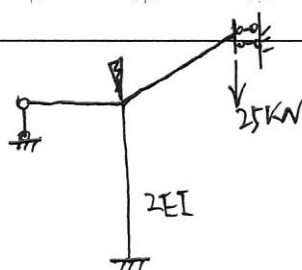
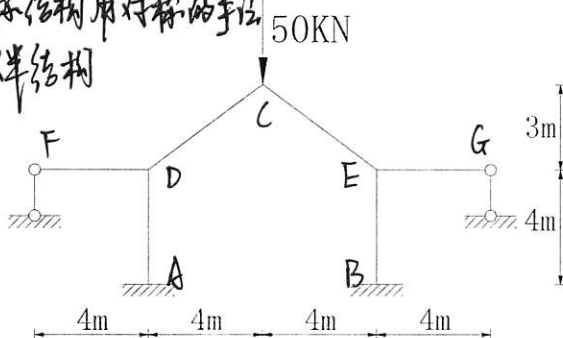
$$\{P_E\} = -\{F_P\} = \begin{pmatrix} 42 \\ \frac{203}{6} \\ \frac{175}{6} \end{pmatrix}$$

七、求图示具有刚性段的梁的自振频率及主振型, 并画出主振型形状, 杆件分布质量不计。(12分)

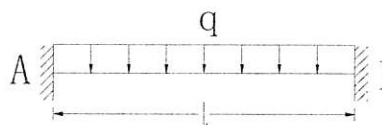


八、用位移法作图示结构的M图, 设两竖柱抗弯刚度为 $2EI$, 其余各杆为 EI 。(13分)


对称结构用对称的方法
取半结构



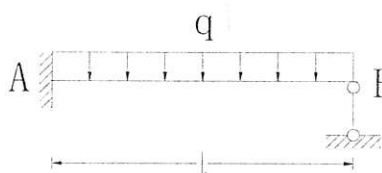
参考资料:



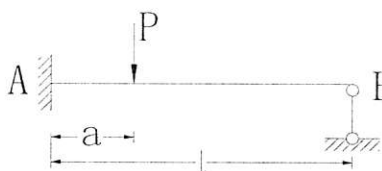
$$M_{AB} = -M_{BA} = -\frac{1}{12} q l^2$$



$$M_{AB} = -\frac{Pa(l-a)^2}{l^2}, \quad M_{BA} = \frac{Pa^2(l-a)}{l^2}$$



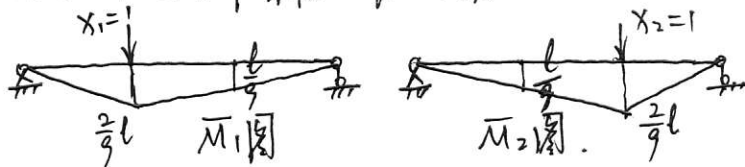
$$M_{AB} = -\frac{q l^2}{8}$$



$$M_{AB} = -\frac{Pb(l^2 - b^2)}{2l^2}, \quad b = l - a$$

$$a = b = \frac{l}{2} \text{ 时 } M_{AB} = -\frac{P \cdot \frac{l}{2} \left(l^2 - \frac{l^2}{4} \right)}{2l^2} = -\frac{P \cdot \frac{3}{8} l^3}{2l^2} = -\frac{3Pl}{16}$$

第七题：本题采用柔度法较为简便。



$$\delta_{11} = \delta_{22} = \frac{1}{EI} \left(\frac{1}{2} \times \frac{1}{3} \times \frac{2}{9} l \times \frac{2}{3} \times \frac{2}{9} l + \frac{1}{2} \times \frac{1}{3} \times \frac{1}{9} \times \frac{2}{3} \times \frac{1}{9} \right) = \frac{5l^3}{729EI}$$

$$\delta_{12} = \delta_{21} = \frac{1}{EI} \left(\frac{1}{2} \times \frac{1}{3} \times \frac{2}{9} l \times \frac{2}{3} \times \frac{1}{9} \right) \times 2 = \frac{4l^3}{729EI}$$

振动方程：

$$\begin{cases} -m\ddot{y}_1 \delta_{11} - m\ddot{y}_2 \delta_{12} = y_1 \\ -m\ddot{y}_1 \delta_{21} - m\ddot{y}_2 \delta_{22} = y_2 \end{cases}$$

设 $y_1 = A_1 \sin(\omega t + \alpha)$
 $y_2 = A_2 \sin(\omega t + \alpha)$
 (圆频率时不计刚性段，请思考为什么？用位移计算公式的原理解释之)

$$\begin{cases} A_1 \omega^2 m \delta_{11} + A_2 \omega^2 m \delta_{12} = A_1 \\ A_1 \omega^2 m \delta_{21} + A_2 \omega^2 m \delta_{22} = A_2 \end{cases}$$

$$\begin{cases} (\omega^2 m \delta_{11} - 1) A_1 + \omega^2 m \delta_{12} A_2 = 0 \quad \dots (1) \\ \omega^2 m \delta_{21} A_1 + (\omega^2 m \delta_{22} - 1) A_2 = 0 \quad \dots (2) \end{cases}$$

A_1, A_2 不全等=0, \therefore 特征行列式

$$\begin{vmatrix} \omega^2 m \delta_{11} - 1 & \omega^2 m \delta_{12} \\ \omega^2 m \delta_{21} & \omega^2 m \delta_{22} - 1 \end{vmatrix} = 0$$

由此解得： $\omega_1^2 = \frac{1}{m(\delta_{11} - \delta_{12})}$ $\omega_2^2 = \frac{1}{m(\delta_{11} + \delta_{12})}$ 代入到 (1) 式或 (2) 式可得 $\frac{A_{11}}{A_{21}}$ 和 $\frac{A_{12}}{A_{22}}$

第八题：前题：



力法系数和自由项： $\delta_{11} = \frac{1}{6EI} (2 \cdot l \sin \alpha \cdot l \sin \alpha) = \frac{l^3 \sin^2 \alpha}{3EI}$

$$\delta_{12} = \delta_{21} = -\frac{1}{EI} \left(\frac{1}{2} \cdot l \sin \alpha \cdot l \times 1 \right) = -\frac{l^2 \sin \alpha}{2EI}$$

$$\delta_{22} = 1 \cdot l \cdot 1 \cdot \frac{1}{EI} = \frac{l}{EI}$$

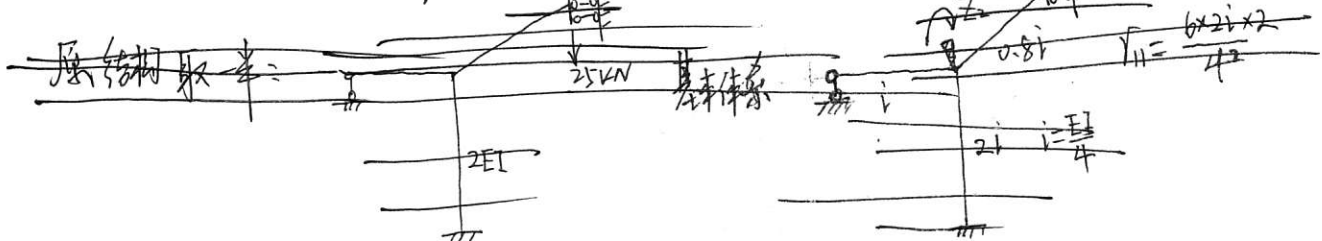
$$\Delta_{1P} = \frac{l}{6EI} (-2l \sin \alpha \cdot P \cos \alpha) = \frac{-Pl^2 \sin \alpha \cos \alpha}{6EI}$$

$$\Delta_{2P} = \frac{1}{EI} \left(\frac{1}{2} \cdot P \cos \alpha \cdot l \cdot 1 \right) = \frac{Pl \cos \alpha}{2EI}$$

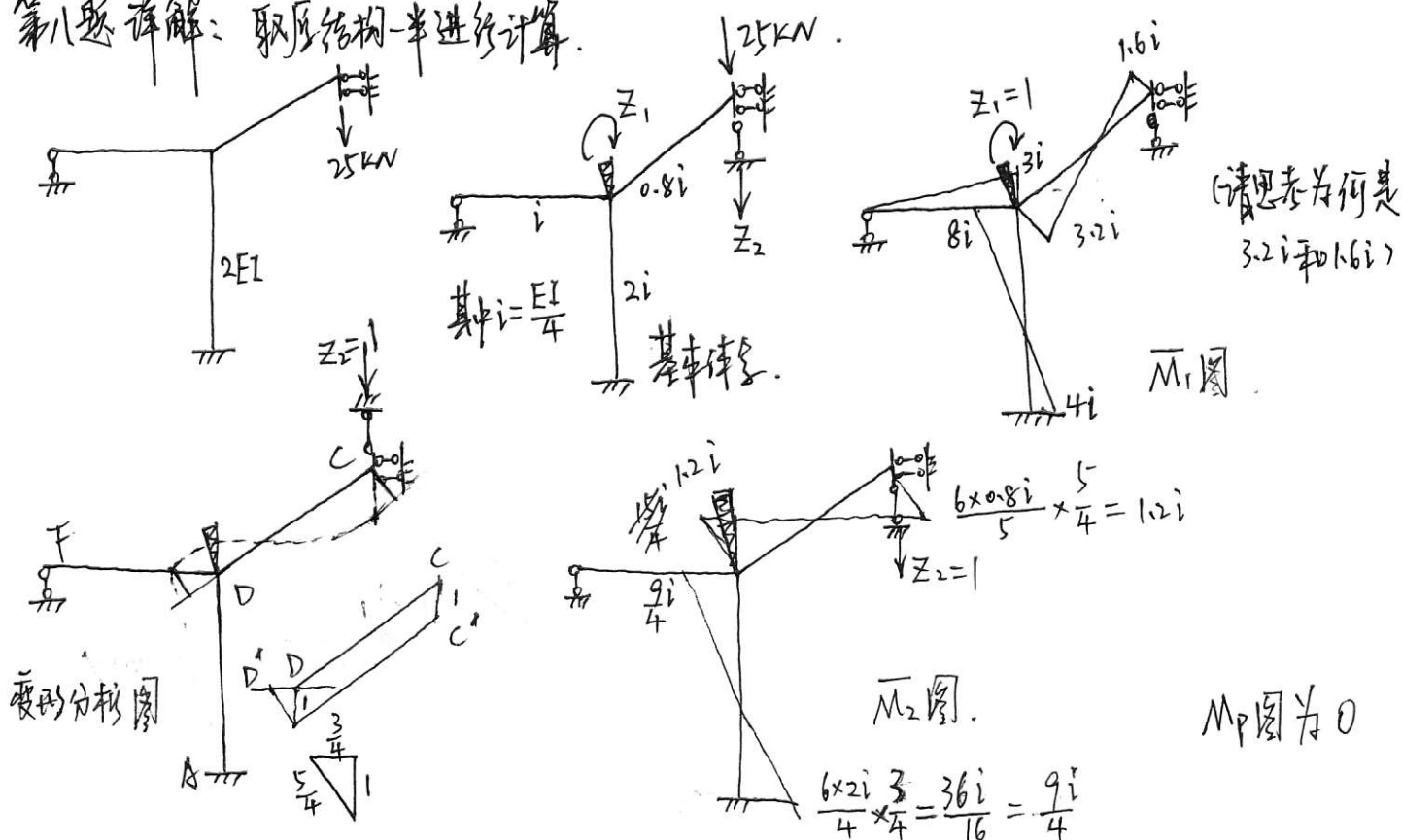
$$\delta_{11} X_1 + \delta_{12} X_2 + \Delta_{1P} = 0 \quad \dots (1) \quad \delta_{21} X_1 + \delta_{22} X_2 + \Delta_{2P} = 0 \quad \dots (2)$$

$$X_1 = P \cos \alpha \quad X_2 = 0 \quad M = X_1 \bar{M}_1 + X_2 \bar{M}_2 + M_P = 0$$

即 (a) 结构在所示荷载 P 作用下无弯矩。直观的理解为杆件两端无位移产生，由位移法也可马上判断 $M=0$ 。



第八题详解：取原结构一半进行计算。



系数和自由项：
$$\gamma_{11} = 3i + 8i + 3.2i = 14.2i$$
$$\gamma_{12} = \gamma_{21} = \frac{9i}{4} - 1.2i = 1.05i$$
$$\gamma_{22} = 1.70625i$$
$$\gamma_{1p} = 0 \quad \gamma_{2p} = -25$$

位移法典型方程：

位移法典型方程：

$$\frac{9i}{4} + 1.2i + \frac{9i}{8} \times 3 = \gamma_{22} \times 4$$

$$\gamma_{22} = 1.70625i$$

$$\begin{cases} \gamma_{11} \cdot Z_1 + \gamma_{12} \cdot Z_2 + \gamma_{1p} = 0 \\ \gamma_{21} \cdot Z_1 + \gamma_{22} \cdot Z_2 + \gamma_{2p} = 0 \end{cases}$$

$$\begin{cases} Z_1 = -1.135i \\ Z_2 = 15.351i \end{cases}$$

$$M = Z_1 \cdot \bar{M}_1 + Z_2 \cdot \bar{M}_2 + \frac{M_p}{=0}$$
 可作得 M 图。



申明：此题答案正确与否有待探讨，
但分析过程无误。

非直角杆件体系弯矩分析是难点：

